



Misure a contatto con evolventimetri e CMM applicate alla misura di ingranaggi ed elementi complessi

Outline

- the project "DriveTrain"
- case study: a large ring segment
- case study: gear roughness measurements







The EMRP is jointly funded by the EMRP participating countries within EURAMET and the European Union



Traceable Measurement of Drivetrain Components for Renewable Energy

Ensure Reliable Quality Assurance for Less Failures and Enhanced Efficiency

www.ptb.de/emrp/drivetrain.html

- 7 national metrology institutes (NMI, coord. PTB);
- 3 universities;
- 4 partners from industry;
- 19 collaborators/stakeholders
- 09/2014 08/2017 Run-time



Technical / Economical needs



- \checkmark EU industry doubling wind energy systems (WES) every 3-4 years
- \checkmark maximum power per WES is now 7 MW, 20 MW is already predicted
- \checkmark high operation costs





Technical needs

Study the metrological needs for

- \checkmark shafts up to 3 m in length and Ø 1 m
- \checkmark large bearings up to Ø 2 m
- \checkmark internal and external gears up to Ø 3 m
- ✓ brakes up to Ø 1 m



DriveTrain





- Network of European NMIs
- Metrological infrastructure for drivetrain components
- Product-like measurement standards for size, form, waviness, and roughness
- Determination of the measurement uncertainty in typical harsh environmental conditions in industry
- ✓ compliance with ISO 14253 to be established

• Equipment

- ✓ Modern measurement systems (GMMs and CMMs)
- Sophisticated software and algorithms
- Certified standards





measurement S U **e g** at $\overline{\mathbf{m}}$ and 20

The aim is to provide candidate 2D and 3D solutions for drivetrain component roughness waviness and form characterisation compliant with GPS methodology

Designed and manufactured

test gears for evaluating characterisation parameters; CAD model links

Issued

a platform for gear profile evaluation; evaluation of ISO 1328-1 filter specification; report on existing 3D gear measurement methods; Gear 2D harmonic profile content; Gear flank and root roughness and waviness dataset; test gear measurement strategy recommendations; a good practice guide (draft) for shaft measurements; metrology strategies for braking system components;

Measured test gear samples; gear replication method assessment;









emen novel The aim is to research and develop measurement standards and calibration procedures for establishing traceability and estimate measurement uncertainty for drivetrain components.

Designed and manufactured

involute artefacts; a large ring with thermo sensors; a segment of a large ring; a novel interferometric step gauge and test gears

Measured selected gear samples







virtual measuring process

Implementation



The aim is to develop a virtual measuring process to include all the significant uncertainty contributions from the workpiece, environment, measuring strategies and measuring instrument.

Development

Model (FEM, Kriging theory) for simulation of representative large drivetrain components allowing estimation of temperature, gravity, surface form and roughness influence to numerical value of measurement uncertainty contribution.

Application

Input data for model: 3D geometry, environmental condition, coefficient of thermal expansion, etc. Simulation: geometrical changes, acclimatization time, deformation. Result: estimation of measurement uncertainty contribution.

Validation

Validation of virtual measurement process model using the experimental measurement. Case studies of comparison of the values obtained by simulation with the measured ones.









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The aim is to test the developed measurement standards in industry and to establish and quantify the key additional sources of uncertainty that influence industrial measurement capability, with particular reference to environmental effects.

Protocols

Measurement procedures are under discussion with industrial partners and other industrial users to find optimal and cost effective realisations for practical purposes.

Established measurement strategies have been transfered to large scale metrology and are under test.

The results are discussed with industry to analyse their performance.

Training on measurement of gears









feasibility of using a ring segment primarily to quantify the influence of the workpiece surface and form, which are generally the largest sources of noise and vibrations of critical rolling surface;

measurements of form, sizes and texture of cylindrical- and torus-shape surfaces are needed to qualify raceway conformity and geometries of large bearings cups.







Measurands

- Two (coaxial) geometrical features:
 - A cylinder
 - A torus
- Intrinsic parameters
 - Radius of the cylinder
 - Torus
 - Radius of the ring
 - Radius of the tube



- Location parameters (axes)
 - Orientation
 - Location





- The measurand features are very partial
 - Cylinder: about its axis
 - Torus: about its axis and about its ring
- Mathematically, a partial element poses an illconditioned problem
 - The CMM errors are magnified
 - Extreme care in compensating the CMM errors
 - Non trivial uncertainty evaluation (by Monte Carlo)







What affects the measurement most?



Comparisons with three calibrated standard to correct as much of these errors as possible







Comparisons







Experimental set up









results

- significantly different in the tangential and radial directions
 - the tangential is much more affected by the partiality of the features
 - this affects both the measurement and the machining
- uncertainty evaluation still to do

Cylinder					
Radius (DfN)	l) 0.16 mm				
Form	9.2 μm				
Torus					
Tube radius (DfN)	idius (DfN) 0.58 mm				
Overall radius (DfN)	-0.10 mm				
Coaxiality torus - cylinder					
Radial	6.15 μm				
Tangential	0.51 mm				
Angle torus - cylinder axes					
Radial plane	-69 µrad				
Tangential plane	-0.43 mrad				
DfN = Deviation from nominal					

A Balsamo, R Frizza, GB Picotto, D Corona, *Design, manufacturing and calibration of a large ring segment,* Proc. EUSPEN2016, Nottingham, UK, May 2016







motivation

- surface texture as a source of vibration generation and wearing during normal gear operation;
- relation between roughness and micropitting;
- selection of roughness parameters for the evaluation of gear texture;
- prediction of gear noise from roughness analysis.



$gear \rightarrow waviness/roughness$



2D tooth profiles were taken from three helical gears of different surface finish



hobbed



ground



lapped

sample gear and gear teeth provided by MDM and NCL



$gear \rightarrow waviness/roughness$



PSD graphs _ involute path

The graphs are truncated at the cutoff frequency $(1/\lambda c)$ on the X-axes and at 10⁻⁶ amplitude on the Y-axes:

Straight lines represent an empirical threshold level of energy, 95% of the whole area below the PSD.

Mean of the PSDs calculated from 20 profiles taken on parallel scan paths of 5 mm length along the **involute** direction



(red) hobbed, (green) ground, and (blue) lapped.



$gear \rightarrow waviness/roughness$



PSD graphs _ helix path

The graphs are truncated at the cutoff frequency $(1/\lambda c)$ on the X-axes and at 10⁻⁶ amplitude on the Y-axes:

Straight lines represent an empirical threshold level of energy, 95% of the whole area below the PSD.

Mean of the PSDs calculated from 10 profiles taken on parallel scan paths of 25 mm length along the <u>helix</u> direction



(red) hobbed, (green) ground, and (blue) lapped.

F. Pollastri, GB Picotto, *Surface texture measurements of gear tooth*, Proc. EUSPEN2016, Nottingham, UK, May 2016



large ring deformation



Motivation

understanding the influence of typical supports and clamping fixtures as used in industry for large rings.

the influence of self-weight of large rings by the deformations along axial and radial directions were calculated by assuming three, four and six support points of two rings :

- diameter 1 m , and 2 m;
- height 115 mm, width 40 mm, AISI 440C;
- horizontal laying on flat supports with a contact area of 20 mm x 40 mm





large ring deformation



Results

Nr of support points	Max axial deformation $\Delta z /\mu m$			
	FreeCAD-Calculix FEM simulation		Comsol FEM simulation	
	Diameter 1 m	Diameter 2 m	Diameter 1 m	Diameter 2 m
3	1,88	27,68	1,88	28,07
4	0,60	7,99	0,53	7,55
6	0,13	1,50	0,12	1,36

Nr of support points	Max radial deformation $\Delta r / \mu m$				
	FreeCAD-Calculix FEM simulation		Comsol FEM simulation		
	Diameter	Diameter	Diameter	Diameter	
	1 m	2 m	1 m	2 m	
3	0,44	3,60	0,59	4,24	
4	0,18	1,37	0,19	1,50	
6	0,05	0,30	0,04	0,33	





Impact



- $\checkmark\,$ Reduction of production and maintenance costs
- ✓ Improvement of the performance of drivetrains
- ✓ Extension of the lifetime of renewable energy systems
- ✓ Reduction of scrap
- $\checkmark\,$ Reworking of expensive large scale workpieces on shop floor
- FEA based prediction of failure modes and functional performance of drivetrain components
- Knowledge transfer and service for other fields e.g. ship and aerospace industry
- ✓ International harmonization of trade
- $\checkmark\,$ Competitiveness of EU renewable energy industry
- \checkmark ISO working groups TC 60 (gears) and TC 213 (GPS)
- $\checkmark\,$ National guidelines such as UNI, BSI, DIN and VDI
- ✓ Best practice guides

Metrologia ingranaggi ed elementi complessi, 6 Marzo 2017, MESAP, UI, Torino

Indirect

Basal





Thanks for your attention!

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